

# Final Design Review and Project Proposal

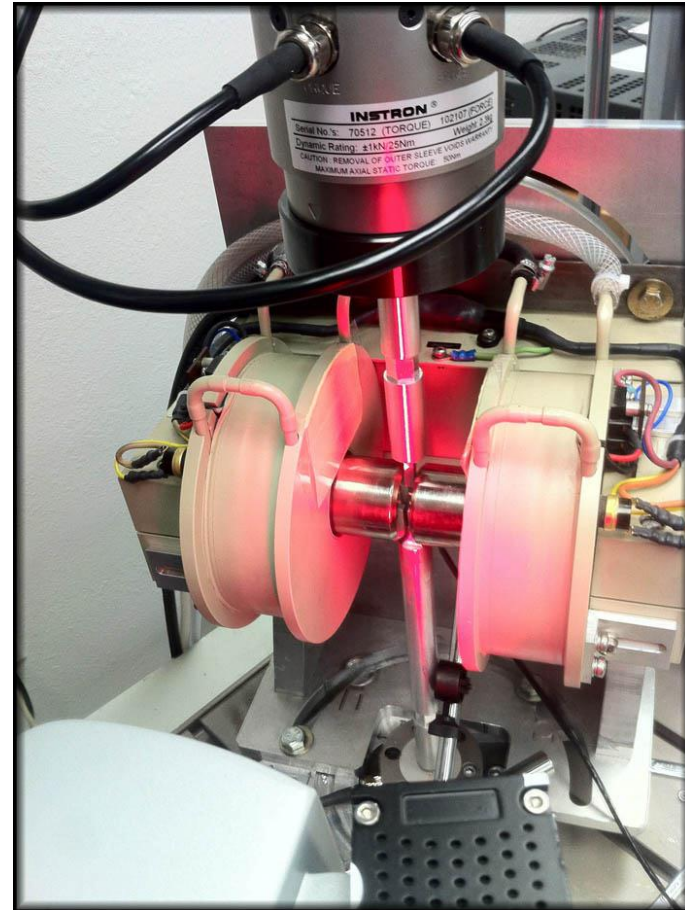
November 27, 2012

**Matt Garcia, Randy Jackson, Jeremy Mountain, Qian Tong, Hui Yao**

**College of Engineering, Forestry, and Natural Sciences  
Northern Arizona University**

# Overview

1. Problem Statement
2. Designs
3. Analysis
4. Selected Design
5. Plans for Next Semester
6. Updated Timeline



# Problem Statement

**Need:** *The eccentric loading of the test specimens causes fatigue failure.*

**Goal:** *Design an improved material testing fixture.*

**Constraints:**

1. Specimen size (3 x 3 x 20) mm
2. Exposed Length ( 6 mm)
3. Grips cannot bite into specimen
4. Push rods and grips must be non-magnetic
5. Distance between magnets (10mm)
6. Magnetic Field (0.5 - 1.0 T )
7. Axial Alignment (50  $\mu\text{m}$ )

**Objectives:**

Objectives	Basis for Measurement	Units
Axially Aligned	Distance from Perfect Alginment	$\mu\text{m}$
Tension Compression Testing	Repeated Testing	# of Tests
Damage Specimen	Cost of Specimen Time to Replace	\$\$ / Month
Inexpensive	Machining Cost Material Cost	\$\$

# Quality Function Deployment

		Engineering Requirements						
		Strain	Tension	Compression	Exposed Length	Grip Size	Magnetic Field	Cost
Customer Requirements	Does not break	X	X	X				
	Tension Test		X					
	Axial Loading		X	X		X		
	Inexpensive				X			X
	Fits in Testing Device				X	X		
	Magnetic Field				X		X	
	See Specimen				X	X		
Units		mm/mm	N	N	mm	mm <sup>2</sup>	T	\$\$
		1.2	18	60	6	100	1	TBD
		Engineering Targets						

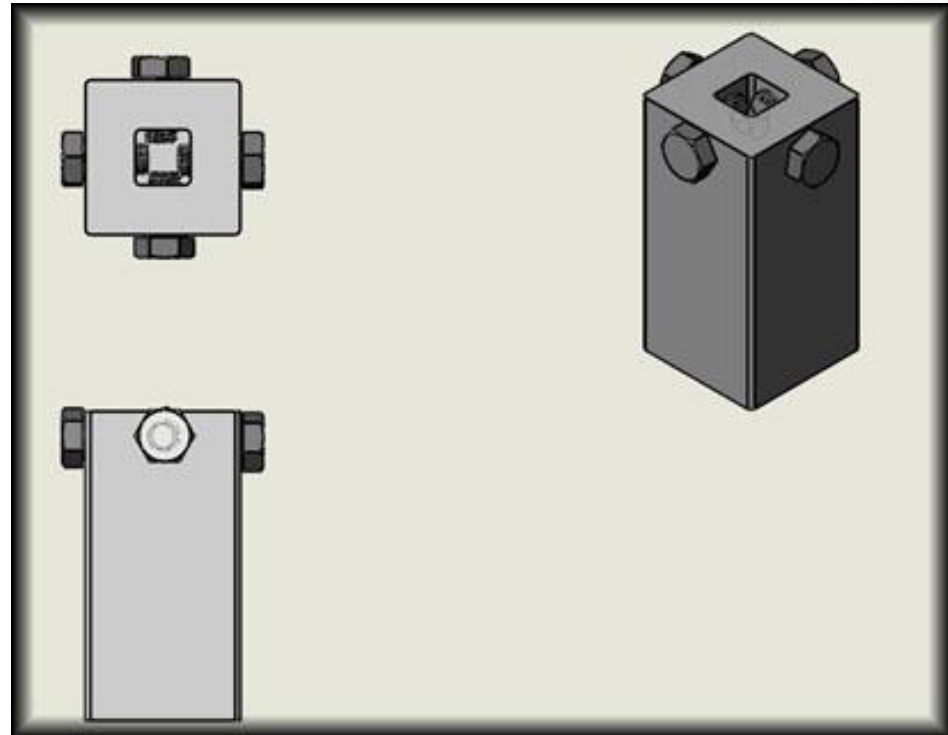
# Generated Tip Design

## Screw Tip

- 4 Set Screws
- Rubber Insert
- Allows Tension Tests

## Problem

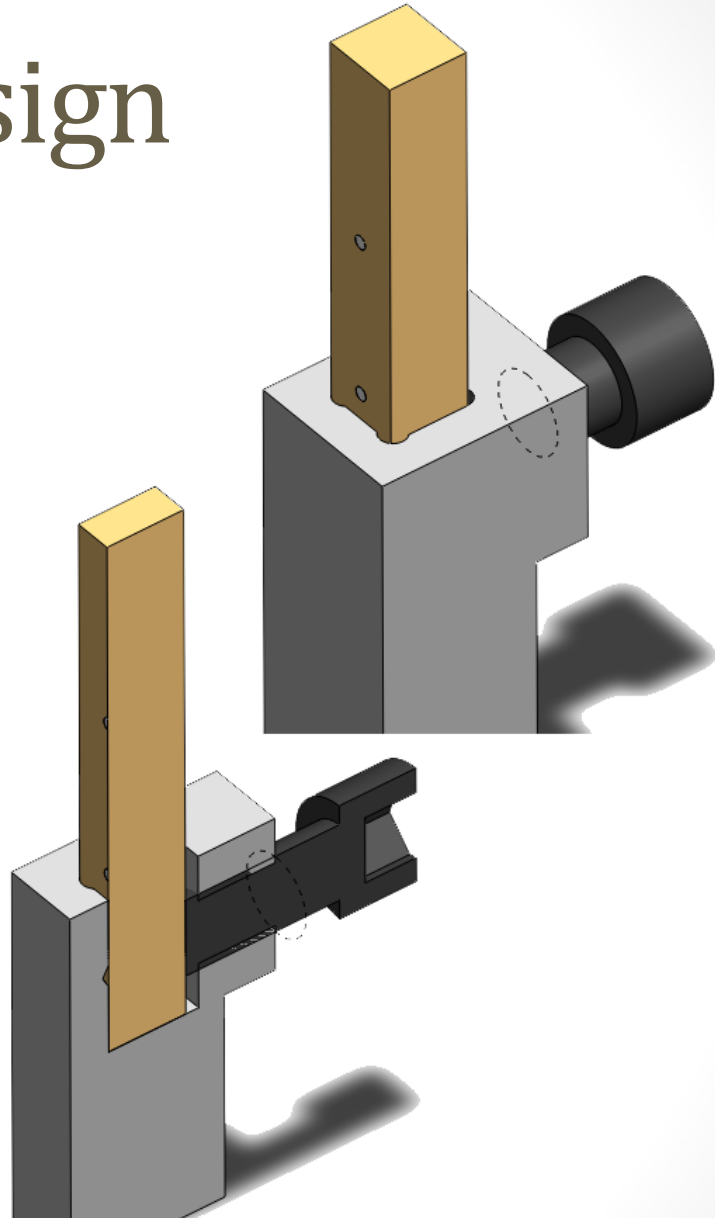
- Axial Alignment



# Selected Tip Design

## Screw Tip

- 1 Set Screw
- Rubber Insert
- Allows Tension Tests
- Axial Alignment



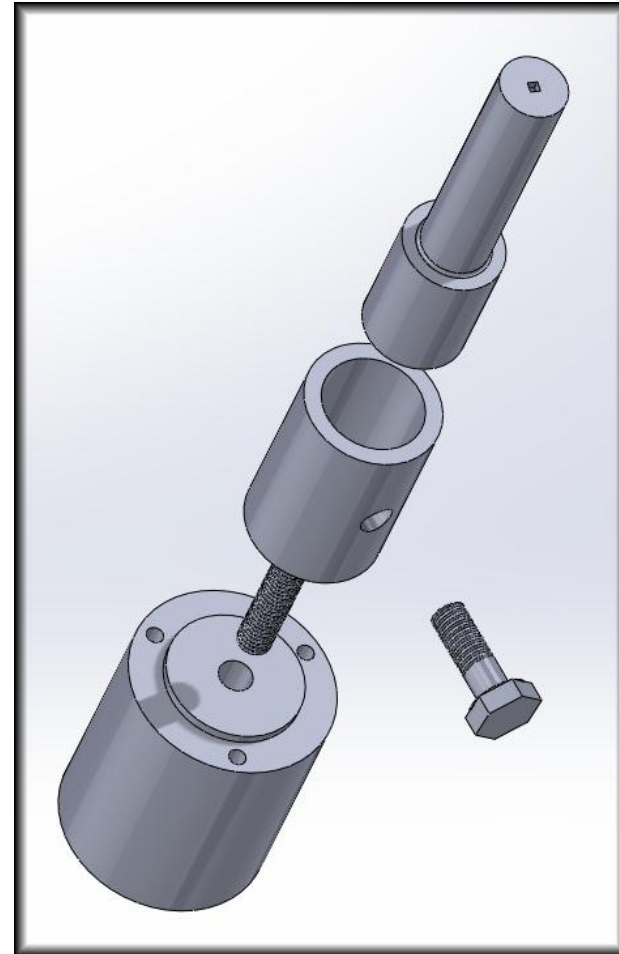
# Generated Base Design

## Base

- Tight Sleeve / Pushrod tolerances
- No Adjustment

## Problem

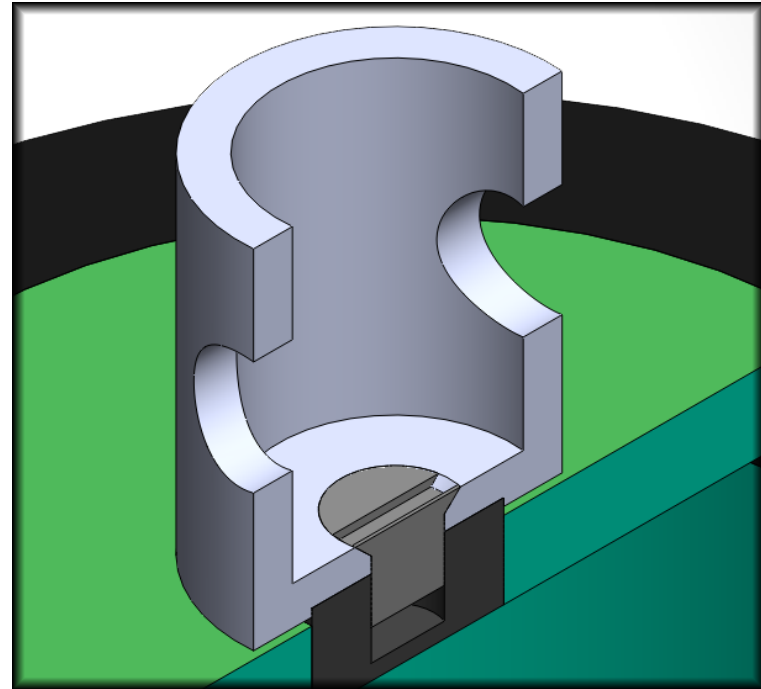
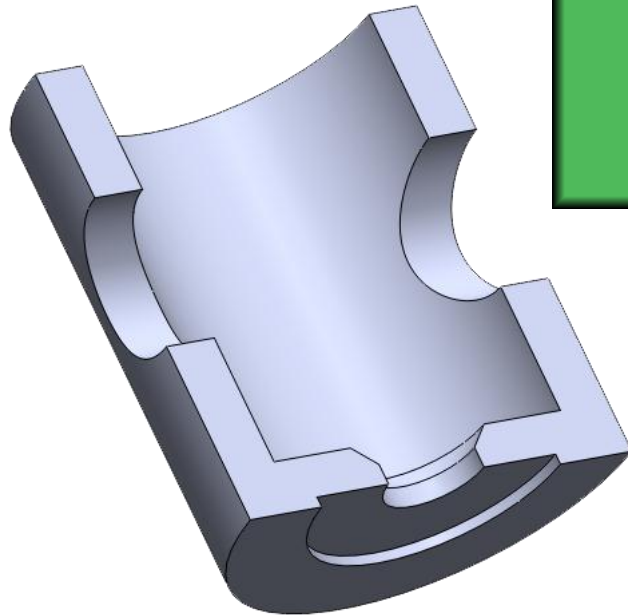
- Inadequate screw tolerance



# Selected Base Design

## Base

- No Alignment Screw
- Uses Existing Alignment
- Tension Tests
- Upper/Lower Fixture





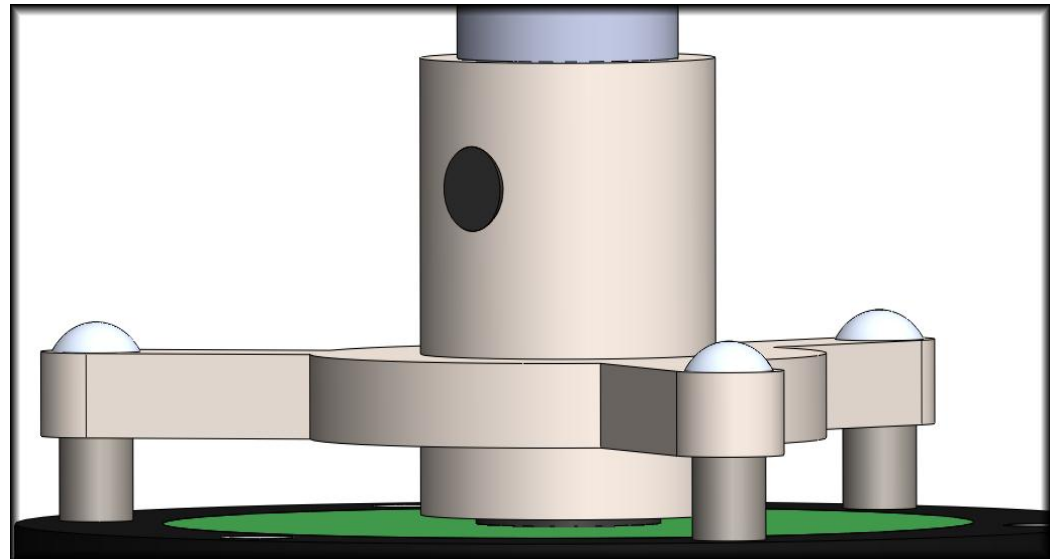
# Generated Base Design

## Base

- 3 Securing Screws
- Securing Pin
- No Adjustment

## Problem

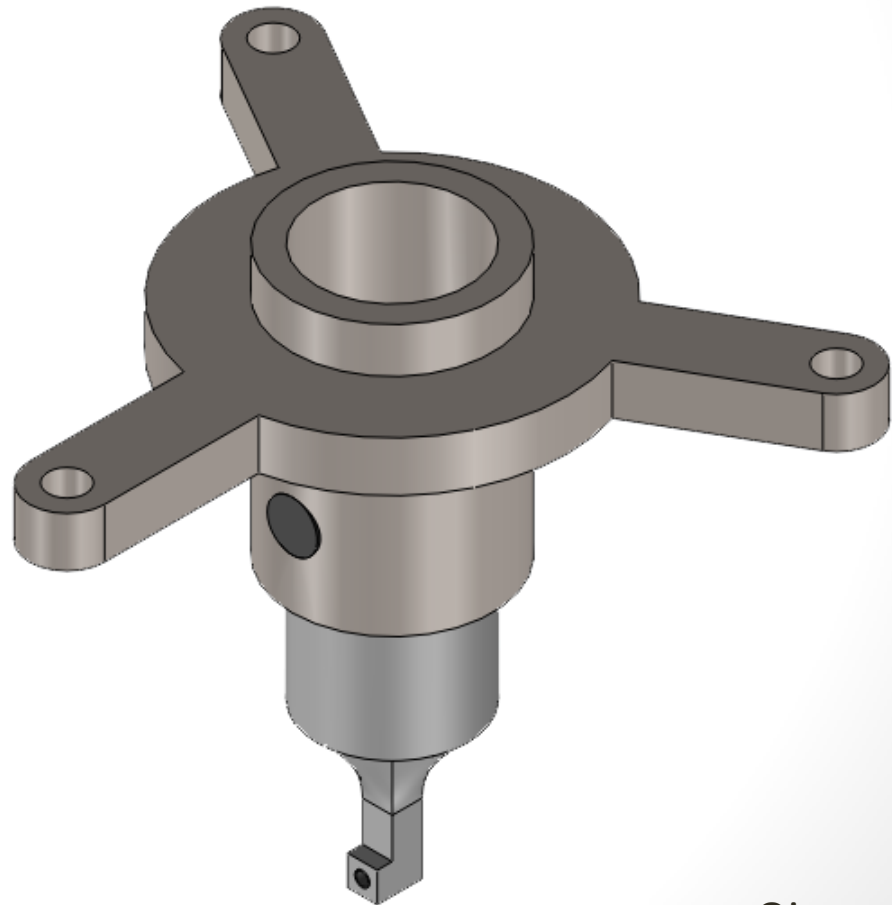
- No Force Analysis Possible



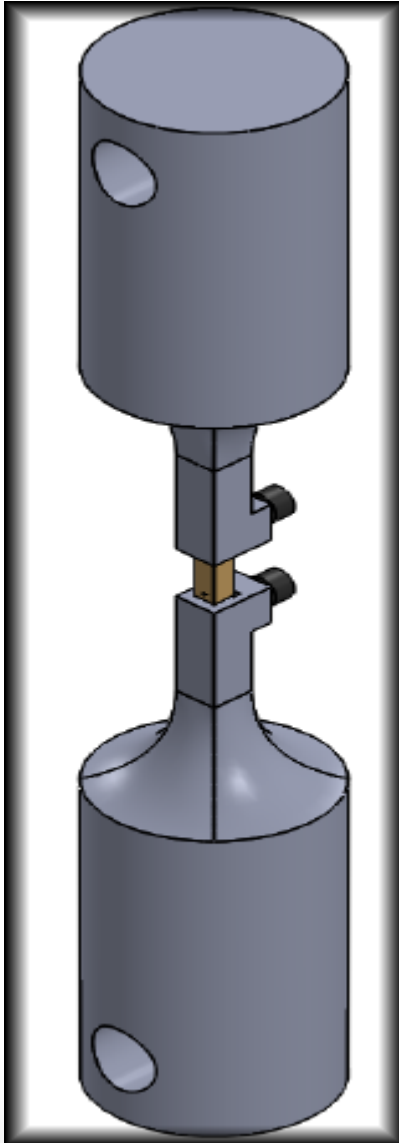
# Modified Base Design

## Alternate Upper Fixture

- No Force Analyzer
- Securing Pin
- Allows Tension Tests
- Non Adjustable



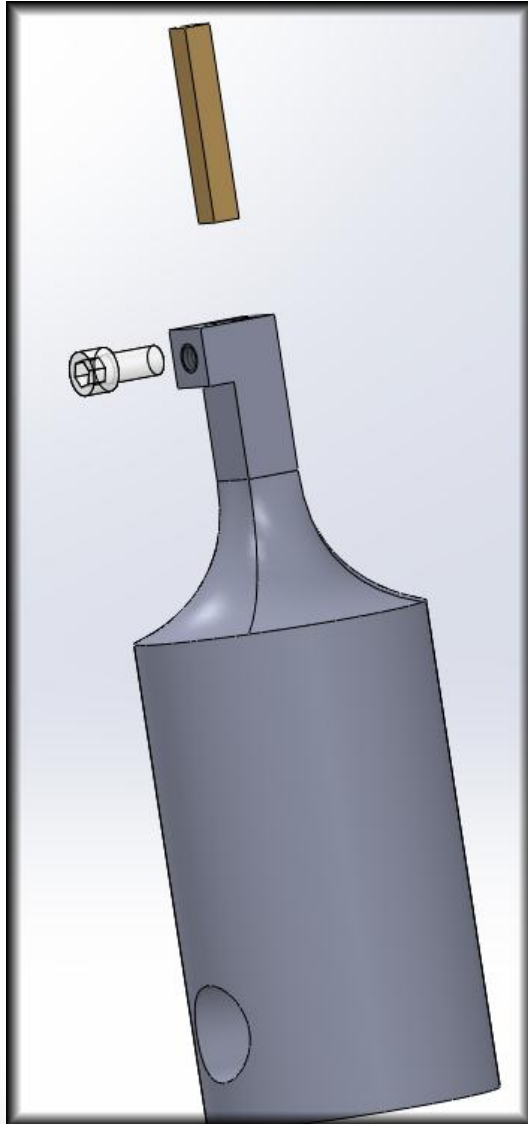
# Material Analysis



## Aluminum 6061 – T6

- A precipitation hardening aluminum alloy.
  - It has good mechanical properties.
  - It is one of the most common alloys of aluminum for general purpose use
- 
- **6061 - T1**
  - **6061 - T3**
  - **6061 - T4**
  - **6061 - T5**

# Material Analysis – Cont.



## Nylon Type 66

- One of the most commonly used polymers.
- Easy and cheap to get.
- Less Yield Strength than aluminum alloy

## Other Considerations

- Brass
- Aluminum
- Rubber Insert

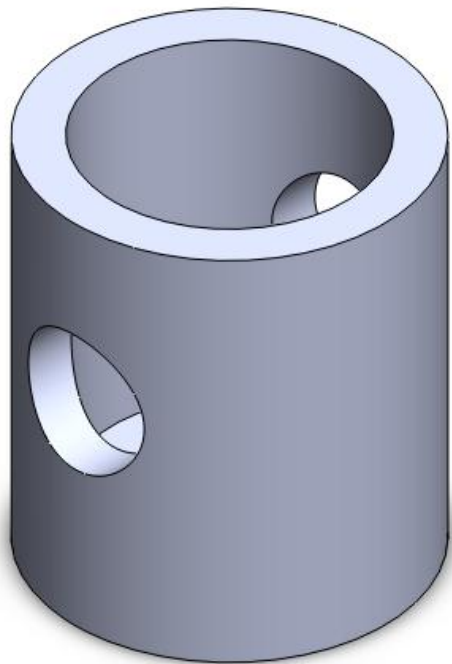
# Compression Analysis

## Smallest Area

<b>Length</b>	3	mm
<b>Width</b>	3	mm
<b>Area</b>	9	mm <sup>2</sup>
	0.000009	m <sup>2</sup>

<b>Force (N)</b>	<b>Stress (N/m<sup>2</sup>)</b>
10	1.111E+06
20	2.222E+06
30	3.333E+06
40	4.444E+06
50	5.556E+06
60	6.667E+06
70	7.778E+06
80	8.889E+06
90	1.000E+07
100	1.111E+07

# Bearing Analysis



	Pin 10mm	Pin 15mm	Pin 20mm	Pin 25mm
Outer Diameter (mm)	Stress (MPa)	Stress (MPa)	Stress (MPa)	Stress (MPa)
31.0	16.00	10.67	8.00	6.40
32.0	8.00	5.33	4.00	3.20
33.0	5.33	3.56	2.67	2.13
34.0	4.00	2.67	2.00	1.60
35.0	3.20	2.13	1.60	1.28
36.0	2.67	1.78	1.33	1.07
37.0	2.29	1.52	1.14	0.91
38.0	2.00	1.33	1.00	0.80
39.0	1.78	1.19	0.89	0.71
40.0	1.60	1.07	0.80	0.64

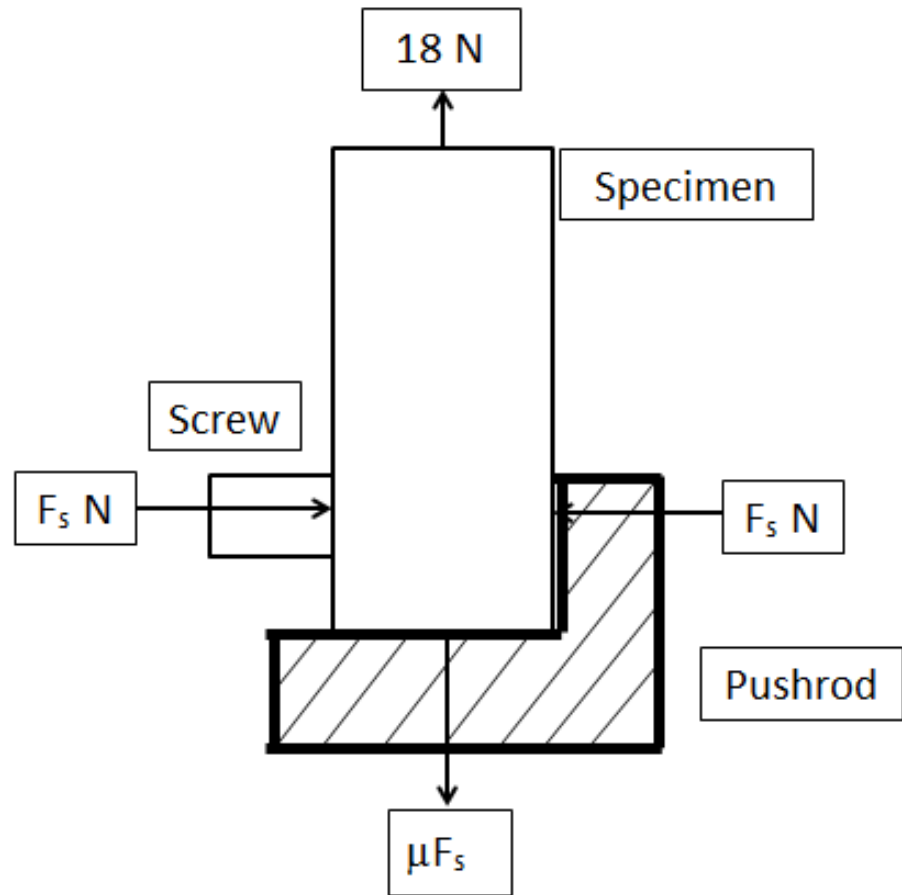
# Screw Analysis

Sum of the forces:

$$\sum F_y = 18\text{N} - \mu F_s = 0$$

$$F_s = \frac{18}{\mu} \text{N}$$

$F_s$ [N]	Friction
120.0	0.2
36.0	0.5
21.2	0.9
15.0	1.2



# Screw Analysis – Cont.

Screw: M3 x 0.5 x 6 mm

Major Diam. D [mm]	Minor Diam. dr [mm]	Thread Engagement Length Le [mm]	Pitch Diam. dp [mm]	Pitch p [mm]	External Shear Area [mm <sup>2</sup> ]	Internal Shear Area [mm <sup>2</sup> ]
3.000	2.385	3.500	2.567	0.500	<b>18.623</b>	<b>32.986</b>

	Nylon Type 66			Brass		
	Yield Str. [MPa]	Force [N]	Coeff. Friction	Yield Str. [MPa]	Force [N]	Coeff. Friction
	45	120	0.15	130	51.43	0.35
<b>External Thread Force to Fail [N]</b>	838.1			2421.0		
<b>Internal Thread Shear to Fail [N]</b>	8081.6			8081.6		



# Cost Analysis

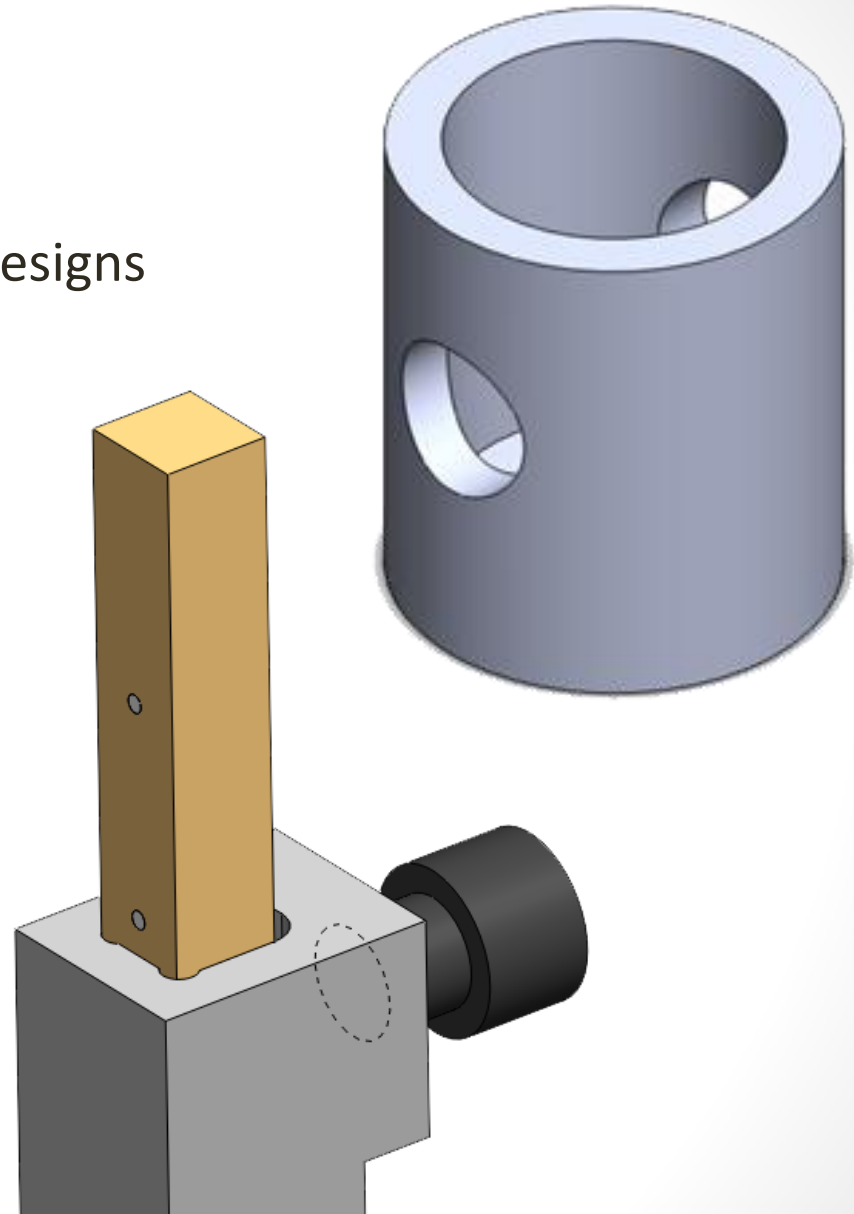
Type	Material	Cost
Main	Aluminum Alloy	0.6-0.9 \$/lb
Screw	Nylon	0.005-0.006 \$/piece

## Other Considerations

- Copper
- Lead
- Magnesium

# The Next Steps

1. Continue with Proposed Designs
2. Regular Customer Input
3. Manufacturing
4. Build Prototype
5. Test Prototype
6. Analysis and Refinement
7. Produce Final Product

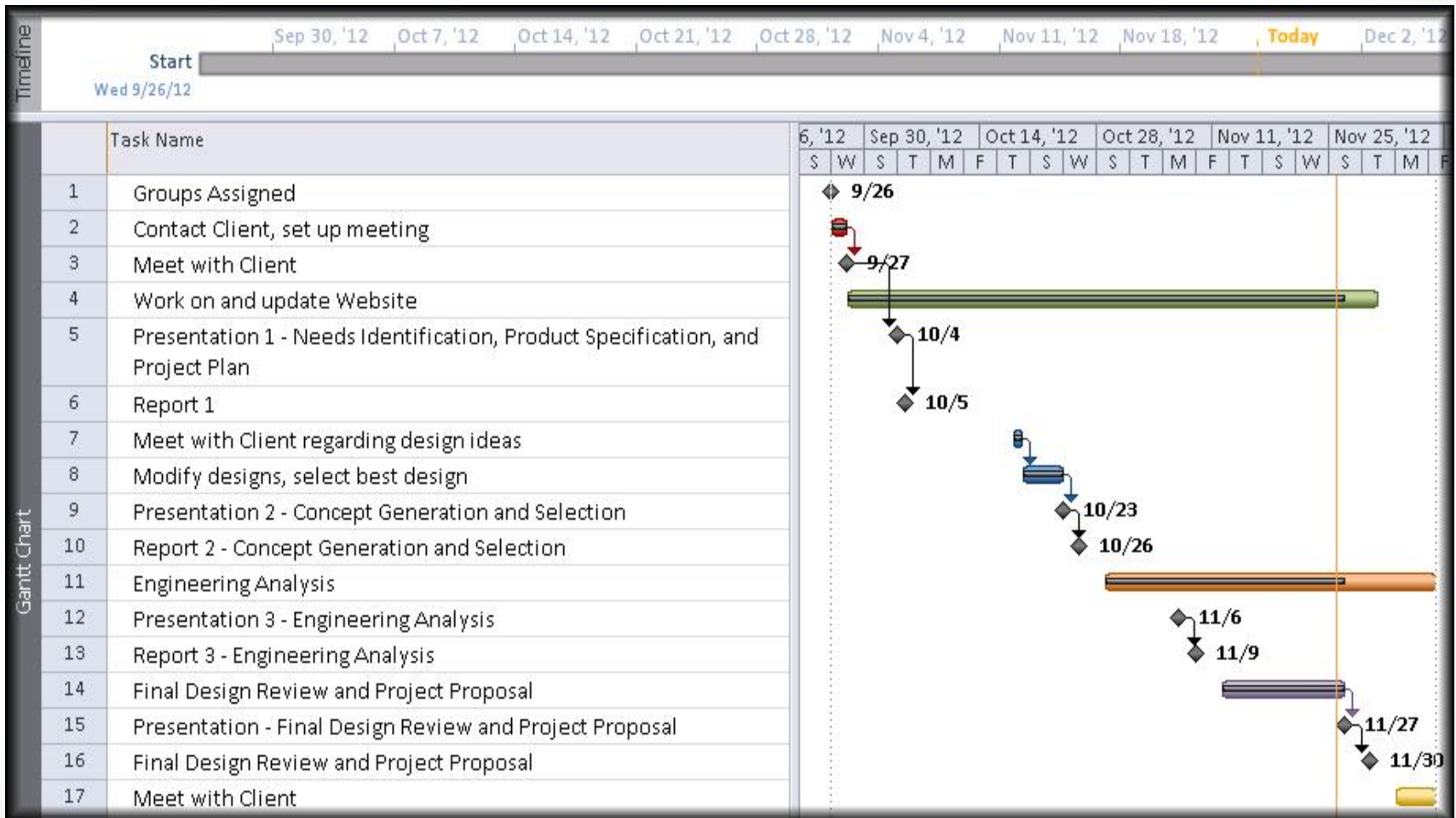


# Conclusion

1. Problem Statement
2. Designs
3. Analysis
4. Future Plans



# Updated Timeline



# References

<http://nau.edu/CEFNS/Engineering/Mechanical/Faculty-Staff/>

<http://www.solidworks.com/>

<http://www.engineershandbook.com/Tables/frictioncoefficients.htm>

<http://www.engineersedge.com>

<http://www.alibaba.com>

<http://www.tcdcinc.com>

[http://www.engineeringtoolbox.com/friction-coefficients-d\\_778.html](http://www.engineeringtoolbox.com/friction-coefficients-d_778.html)

[http://www.youtube.com/watch?v=sPwURRG9\\_Gs](http://www.youtube.com/watch?v=sPwURRG9_Gs)

<http://nau.edu/Research/Feature-Stories/NAU-on-Leading-Edge-of-Smart-Materials-Research/>

Shigley's Mechanical Engineering Design, 9<sup>th</sup> Edition.

Dr. Constantin Ciocanel